

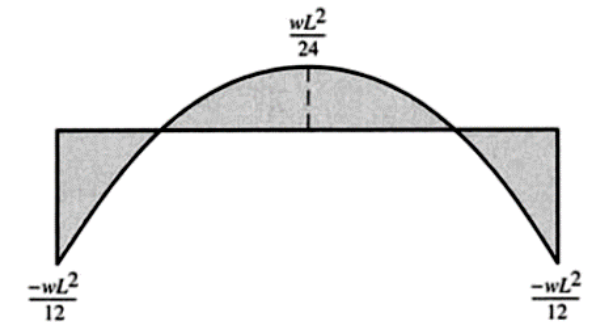
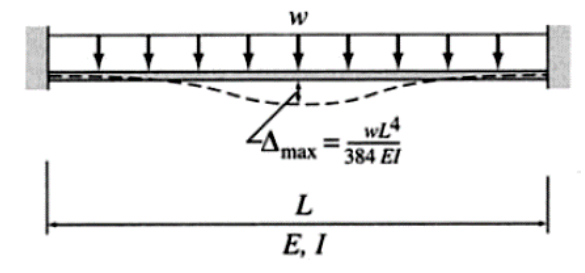
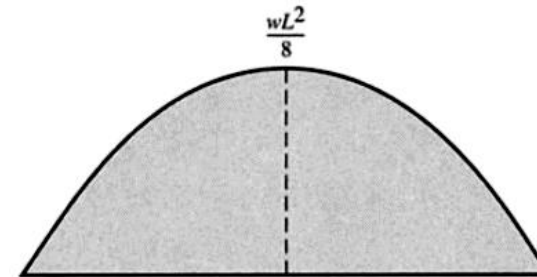
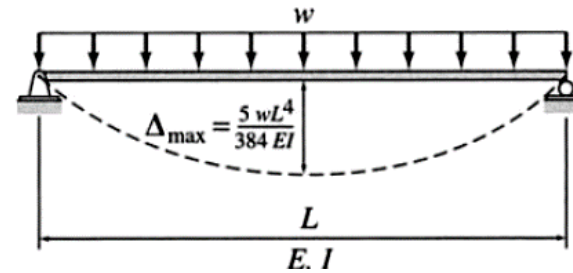


Dr. Khalil M. Qatu

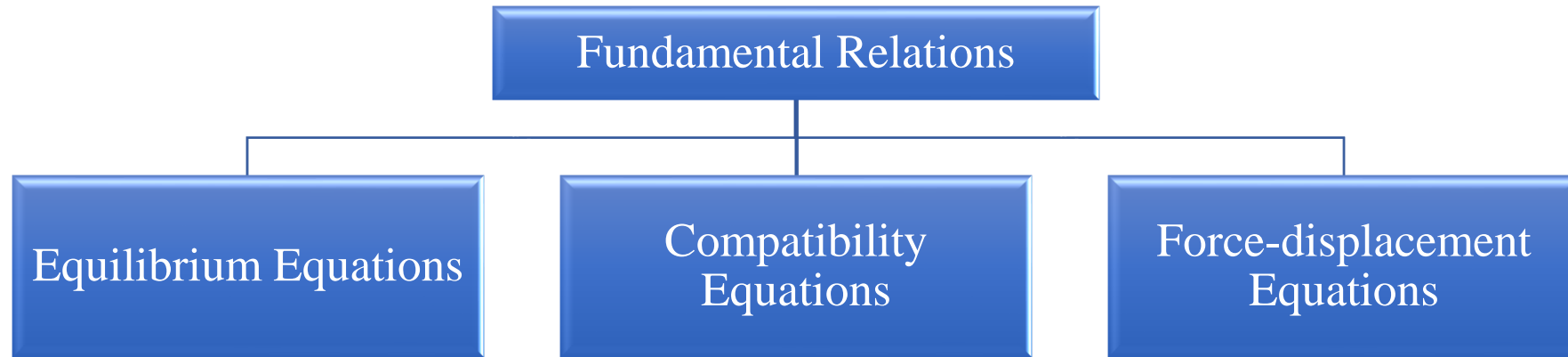
# Analysis of indeterminate structures

# Indeterminate structures

Advantages	Disadvantages
Less stresses , deflections due to loading	More stresses due to temperature and fabrication errors
Cost (member size)	Cost (support condition)
Redundancy	



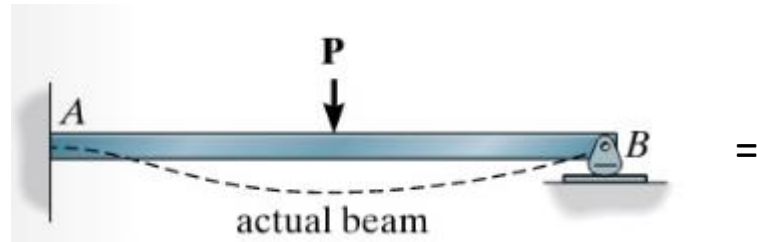
# Analysis of Indeterminate structures



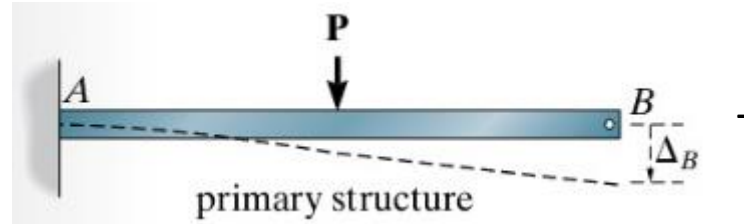
	Unknowns	Equations Used for Solution	Coefficients of the Unknowns
Force Method	Forces	Compatibility and Force Displacement	Flexibility Coefficients
Displacement Method	Displacements	Equilibrium and Force Displacement	Stiffness Coefficients

Force Method  
Indeterminate to 1<sup>st</sup> degree

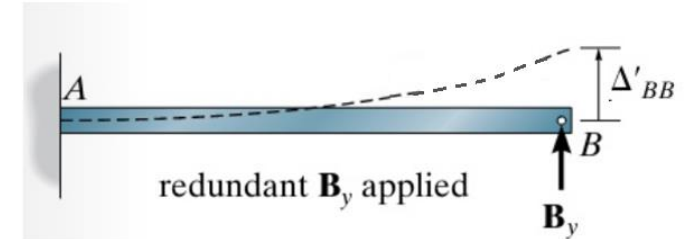
# Force Method



Step 1: Remove a redundant force



Step 2: Reapply redundant force



Step 3: Write Compatibility Equation

$$0 = -\Delta_B + \Delta'_{BB}$$

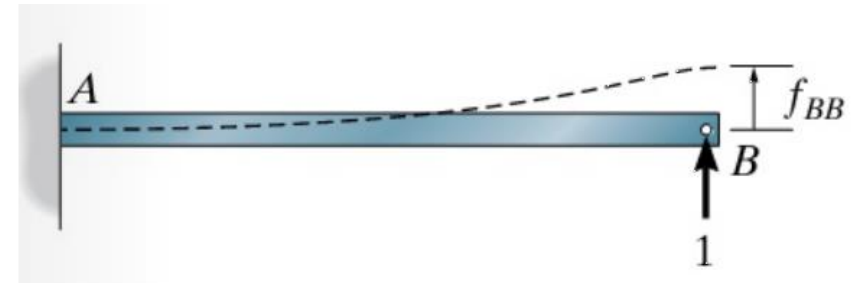
$$0 = -\Delta_B + B_y f_{BB}$$

$$\Delta_B =$$

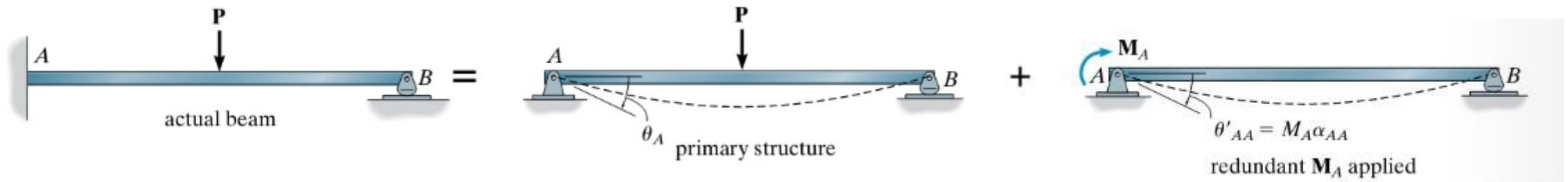
$$\Delta'_{BB} = B_y f_{BB}$$

$$f_{BB} =$$

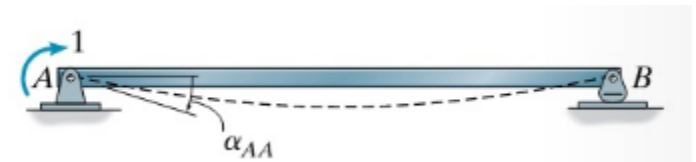
Step 4: Find Displacements



# Force Method



$$0 = \theta_A + M_A \alpha_{AA}$$



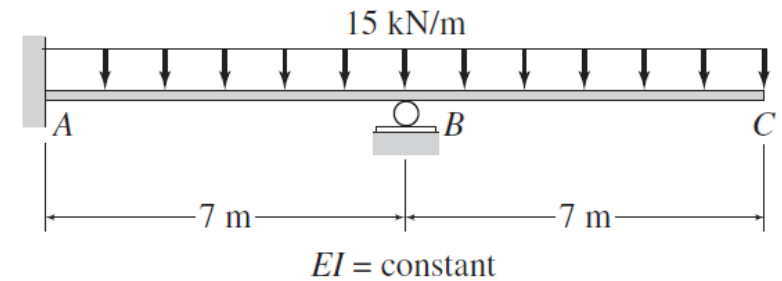
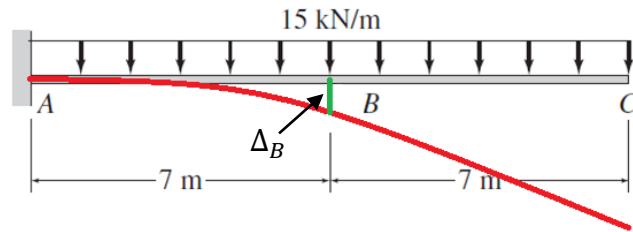
$$\theta_A =$$

$$\alpha_{AA} =$$

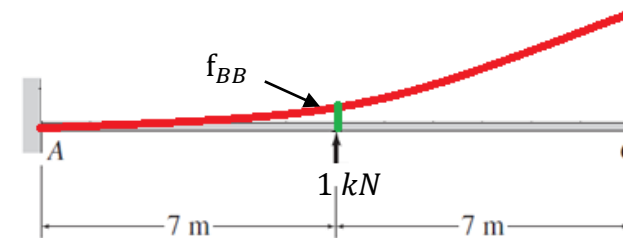
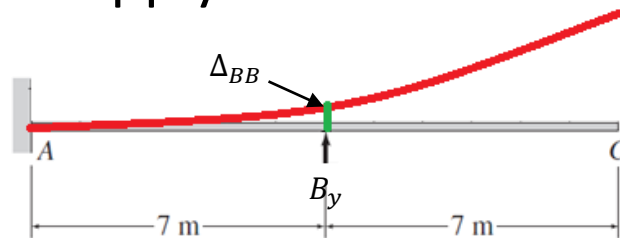
# Beam Example

Draw shear and moment diagrams for the Given Beam

- Step 1: remove redundant support reaction ( $B_y$ )



- Step 2: Reapply redundant reaction ( $B_y$ )

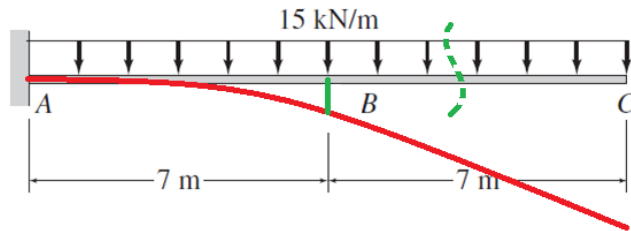


- Step 3: Compatibility equation (deflection upward is positive)

$$-\Delta_B + \Delta_{BB} = 0 \rightarrow -\Delta_B + f_{BB} B_y = 0$$

# Beam Example

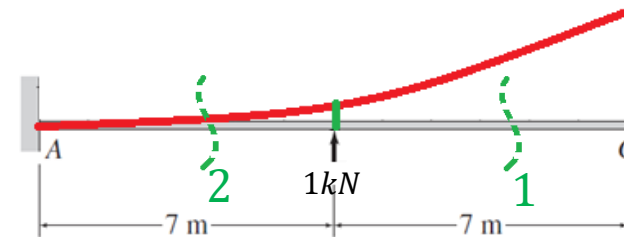
- Step 4: Find deflections ( $\Delta_B, \Delta_{BB}$ )
  - Find  $\Delta_B, f_{BB}$  using virtual work method



Real Structure

$$M = -\frac{15}{2}x^2$$

$0 > x > 14$



Virtual Structure

$$m_1 = 0 \quad 0 < x < 7$$

$$m_2 = x - 7 \quad 7 < x < 14$$

$$f_{BB} = \int_0^7 \frac{mm}{EI} + \int_7^{14} \frac{mm}{EI} = 0 + \int_7^{14} \frac{(x-7)^2}{EI} dx = \frac{1}{EI} \int_7^{14} (x^2 - 14x + 49) dx = \frac{114.3}{EI}$$

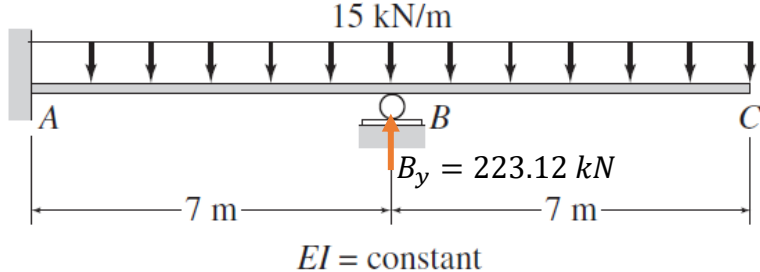
$$\Delta_B = \int_0^7 \frac{Mm}{EI} dx + \int_7^{14} \frac{Mm}{EI} dx = 0 + \int_7^{14} \frac{\left(-\frac{15}{2}x^2\right)(x-7)}{EI} dx = \frac{1}{EI} \int_7^{14} \left(-\frac{15}{2}x^3 + 52.5x^2\right) dx = -\frac{25510}{EI}$$

# Beam Example

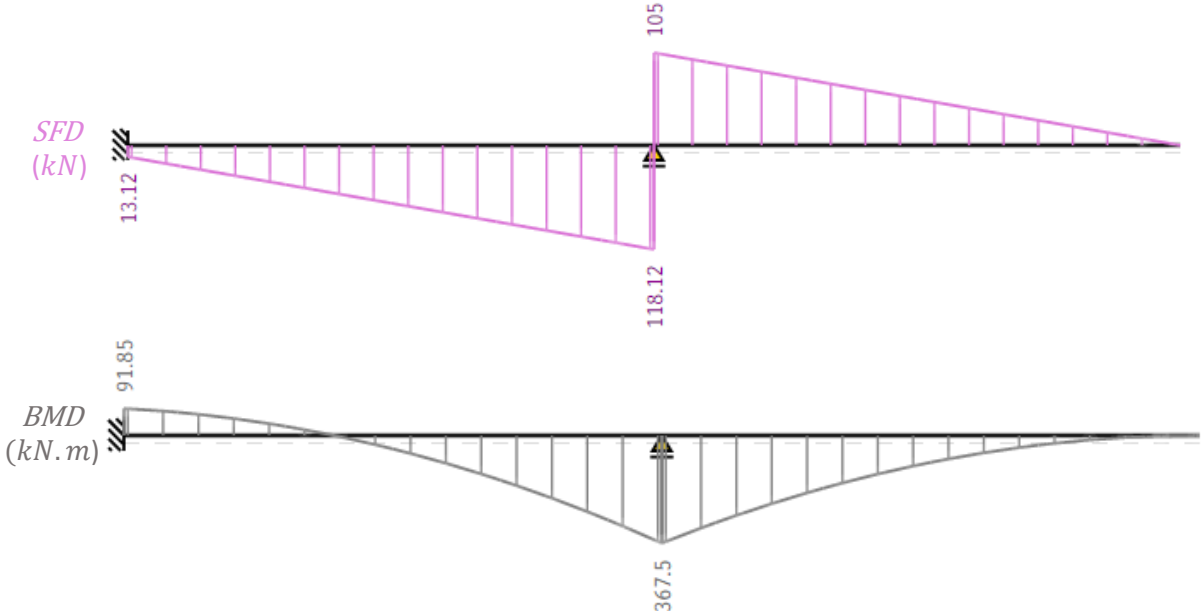
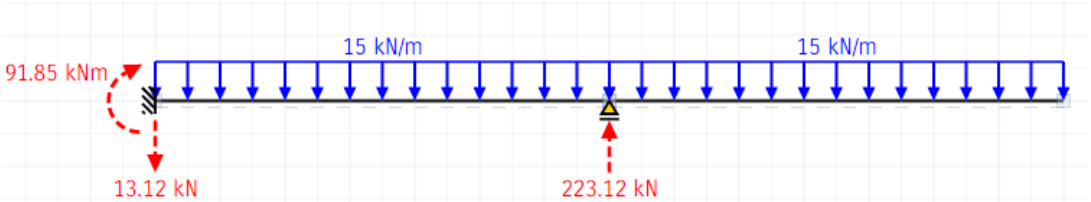
- Step 5: Apply compatibility and solve for  $B_y$

$$\text{compatibility} \rightarrow -\frac{25510.6}{EI} + \frac{114.3}{EI} B_y = 0$$

$$B_y = 223.2 \text{ kN}$$



- Step 6: Find reactions at A and draw shear and moment diagrams

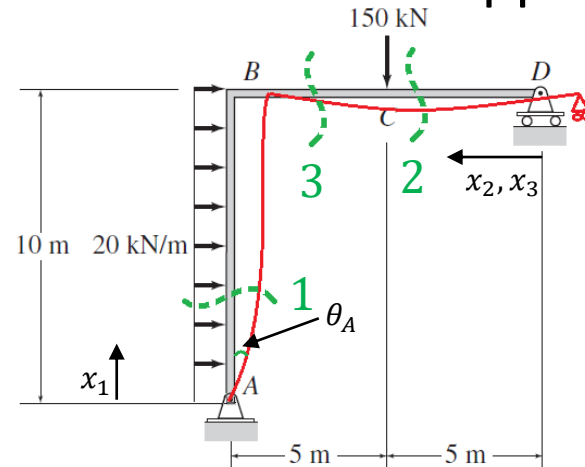




# Frame Example

Draw axial, shear and moment diagrams for the Given Frame

- Step 1: remove redundant support reaction ( $M_A$ )



$$M_1 = 200x - 10x^2$$

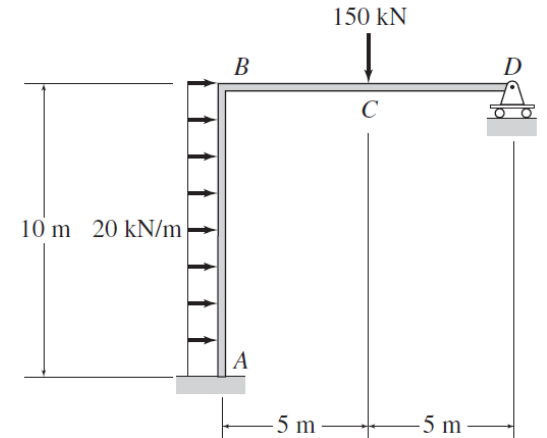
$$0 > x_1 > 10$$

$$M_2 = 175x$$

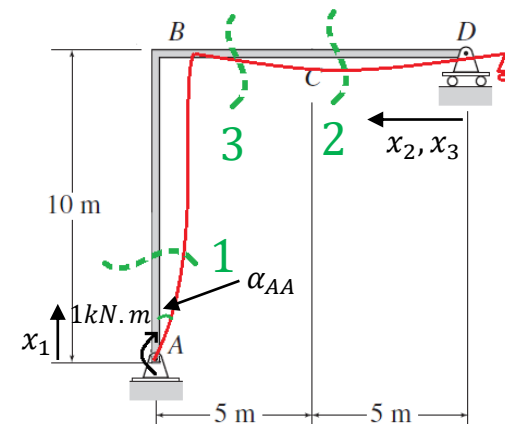
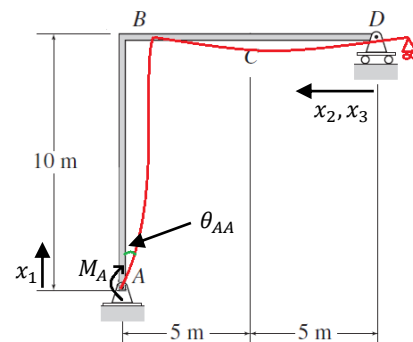
$$0 > x_2 > 5$$

$$M_3 = 25x + 750$$

$$5 > x_2 > 10$$



- Step 2: Reapply redundant reaction ( $M_A$ )



$$m_1 = 1$$

$$0 > x_1 > 10$$

$$m_2 = 0.1x$$

$$0 > x_2 > 5$$

$$m_3 = 0.1x$$

$$5 > x_2 > 10$$

# Frame example

- Step 3: Write compatibility equation (c.w. rotation positive)

$$\theta_A + \theta_{AA} = 0 \rightarrow \theta_A + \alpha_{AA} M_A = 0$$

- Step 4: Find deflections

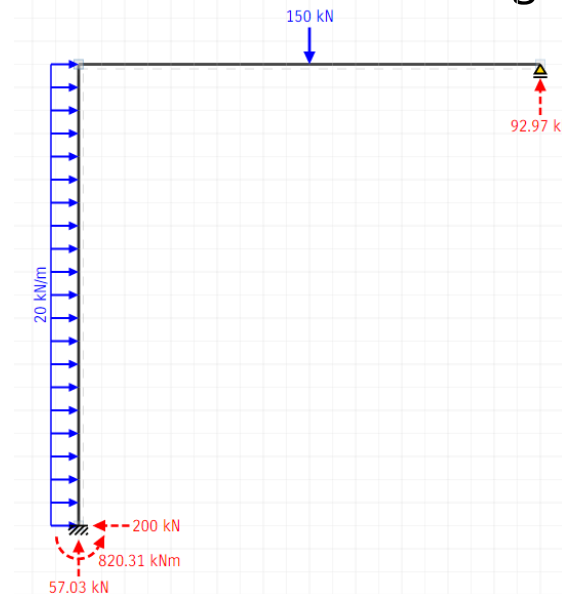
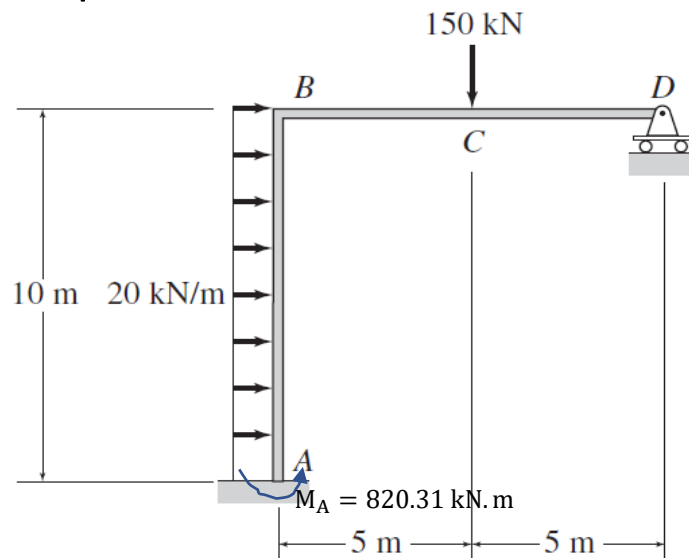
- Find  $\theta_A$  and  $\alpha_{AA}$  using virtual work method

$$\theta_{AA} = \int_0^{10} \frac{M_1 m_1}{EI} dx_1 + \int_0^5 \frac{M_2 m_2}{EI} dx_2 + \int_5^{10} \frac{M_3 m_3}{EI} dx_3 = \frac{10943}{EI}$$

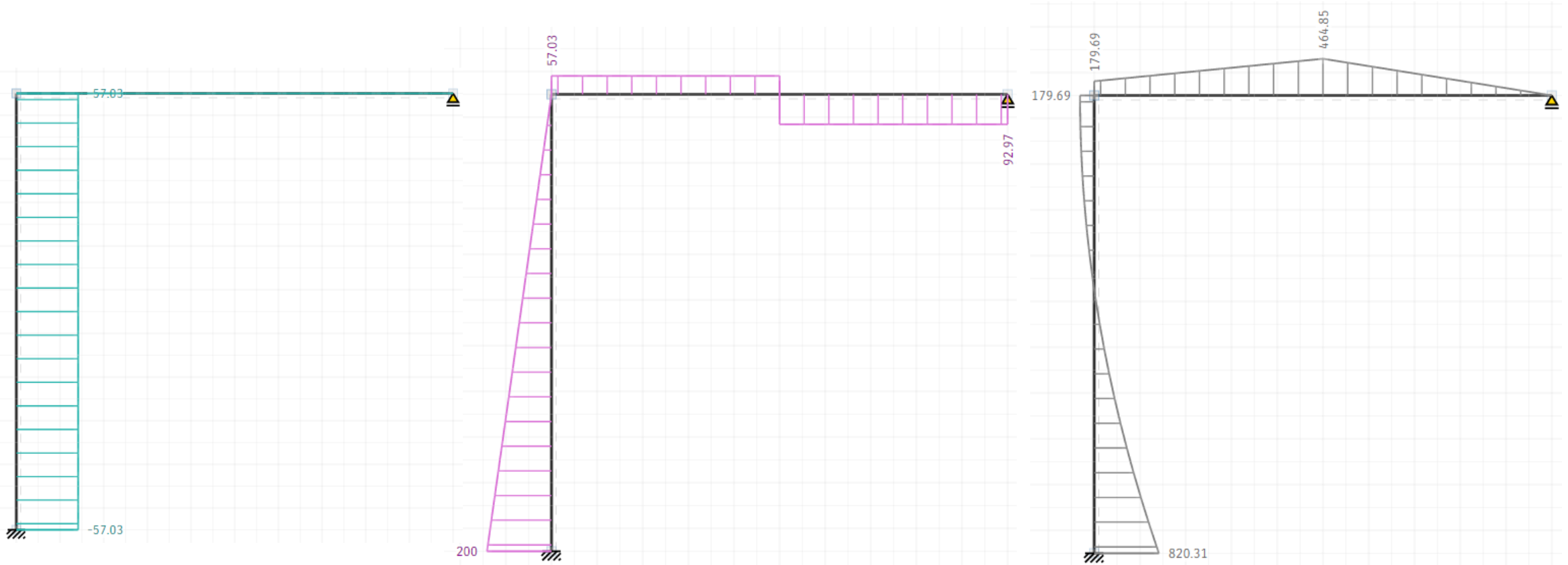
$$\alpha_{AA} = \int_0^{10} \frac{m_1 m_1}{EI} dx_1 + \int_0^5 \frac{m_2 m_2}{EI} dx_2 + \int_5^{10} \frac{m_3 m_3}{EI} dx_3 = \frac{13.34}{EI}$$

- Step 5: Apply compatibility and solve for  $M_A \Rightarrow M_A = -820.31 \text{ kN.m}$  (opposite to assumption)

- Step 6: Find the rest of the reactions and draw shear and moment diagrams



# Frame example



Axial (kN)

SFD (kN)

BMD (kN.m)