

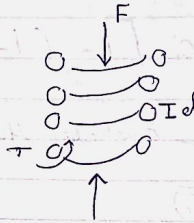
Chapter 10 Mechanical Springs

Wire Spring: \rightarrow always compression forces

Helical springs:

$$\tau_{max} = \frac{F}{A} + \frac{FD}{Jr}$$

$$= \frac{FD}{Jr} + \frac{F}{A}$$



$D =$ mean diameter

$\tau = K_s \frac{8FD}{\pi d^3}$ } K_s : shear correction factor } Not used

$$K_s = \frac{2C+1}{2C}$$

C: Spring index

With Curvature effect:

$$\tau_{max} = K_B \frac{8FD}{\pi d^3}$$

$$K_B = \frac{1C+2}{4C-3}$$

$$C = \frac{D}{d}$$

$$D = OD - d$$

$$K = \frac{d^4 G}{8 D^3 N a}$$

Number of coils (Active)

K: Spring Constant
or Spring rate

- Spring types:
- a) Plain end } right hand
 - b) closed end } \rightarrow \leftarrow
 - c) ground end } left hand
 - d) Plain end }

10.5

Stability

critical deflection:

$$y_{cr} = L_0 C_1 \left[1 - \left(1 - \frac{C_2}{\lambda_{eff}^2} \right)^{\frac{1}{2}} \right]$$

$$C_1 = \frac{E}{2(E-G)}$$

← Elasticity Mod
↘ Rigidity Modulus

$$C_2 = \frac{2\pi^2(E-G)}{2G+E}$$

$$\lambda_{eff} = \frac{\alpha(L_0)}{D}$$

Tables effective slenderness ratio
↘ unstretched length

• stable if. $y < y_{cr} \rightarrow \frac{C_2}{\lambda_{eff}^2} < 1 \rightarrow L_0 < \frac{\pi D}{\alpha} \left[\frac{2(E-G)}{2G+E} \right]^{\frac{1}{2}}$

For steel: (No Buckling)

$$L_0 < 2.63 \frac{D}{\alpha}$$

Spring materials

- Springs materials are usually mixtures of different materials

How to find S_{ut}

$$S_{ut} = \frac{A}{d^m}$$

← Constant **Not Area**

A and m: From tables

S_{sy} = yield shear strength

= Maximum Percent of tensile strength $\times S_{ut}$

Notes for Solving :

concentration \rightarrow Table 10-6 \rightarrow $N_a = \text{total} - 2$ (From table 10-6)

ends are squared $\rightarrow N_a = \text{total} - 2$ (From table 10-6)

no. ^{number} \rightarrow Table A-28 (you are Given the type)

D \rightarrow mean effective diameter **Not outside diameter**

Get d :- $D = \text{outside} - d$

Then get K_b from C ($C = \frac{D}{d}$)

10.9: fatigue loading

Zimmerli Data

peened ← تفتيح لسطح المعدن
unpeened ← بدون تفتيح

shear endurance limit

حارة العزقة

→ Same as welding

Goodman line:

$$\frac{S_a}{S_e} + \frac{S_m}{S_{ut}} = 1 \quad \leftarrow \text{Remember}$$

or

Soderberg:

Note:

Wire Size: d / Free length $= L$ / $N_a = \text{active}$
 $D = \text{out} - d$

• If shear Torsion

$$n_f = \frac{1}{2} \left(\frac{0.67 S_{ut}}{S_{se}} \right) \frac{T_a}{S_{se}} \left[-1 + \sqrt{1 + \frac{4 S_{se}}{0.67 S_{ut}} \frac{T_a}{S_{se}}} \right] \rightarrow \text{Back to Table (6-F)}$$

10.11 Extension Spring

* Different from compression spring

1- There is a hook

2- Pitch = 0 or very small to store max energy

$$\sigma_A = F \left((k)_A \frac{16D}{\pi d^3} + \frac{4}{\pi d^2} \right)$$

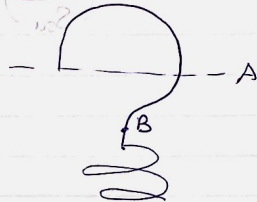
At section A $\rightarrow (k)_A = \frac{4C_1^2 - C_1 - 1}{4C_1(C_1 - 1)}$

$$C_1 = \frac{2r_1}{d}$$

Point B

$$\rightarrow (k)_B = (k)_A \frac{8FD}{\pi d^3}$$

$$\rightarrow (k)_B = \frac{4C_2 - 1}{4C_2 - 1} \cdot C_2 = \frac{2r_2}{d}$$



In the coils, I can use compression spring equations

Static load not included (yielding)

endurance limit $S_e = S_e / 0.577 \rightarrow$ Distortion energy theory

10.12 helical coil torsion springs \rightarrow Not included

$$y = \frac{F - F_i}{(k)}$$

$$\text{closed length} = 2(D - d) + (N_b + 1)d$$