

Heat transfer

- Heat is transferred from **higher** temperature to **lower** temperature until they reach the **same** temperature

Modes of H.T

Conduction

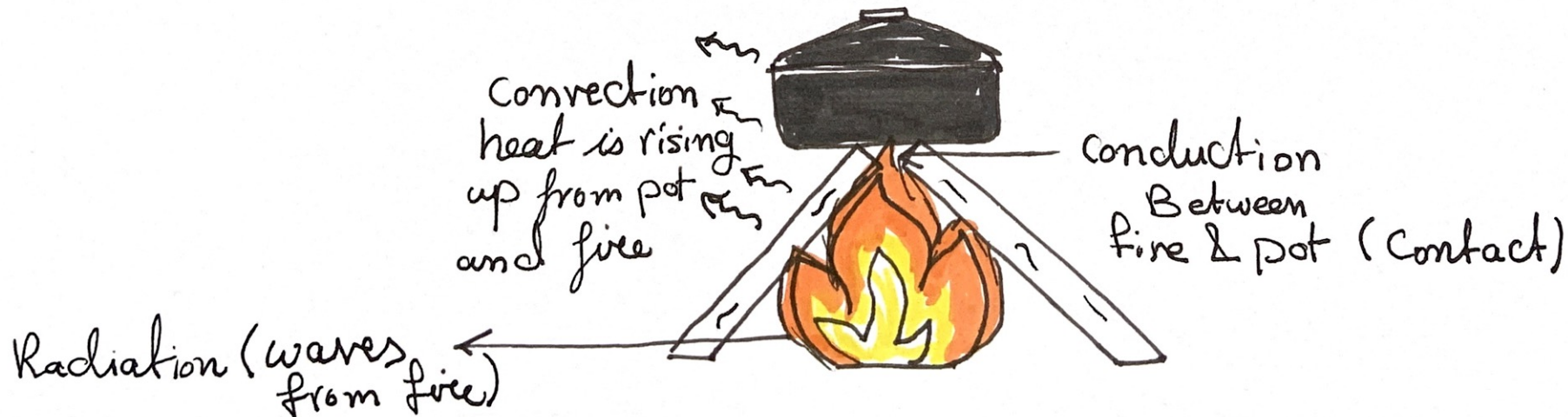
- Direct contact
- material required is **Solids**

Convection

- fluid movement (Density)
- material required is **liquids** and **gases**

Radiation

- wave motion
- **No** material required



1- **Conduction** occurs at direct contact of solids

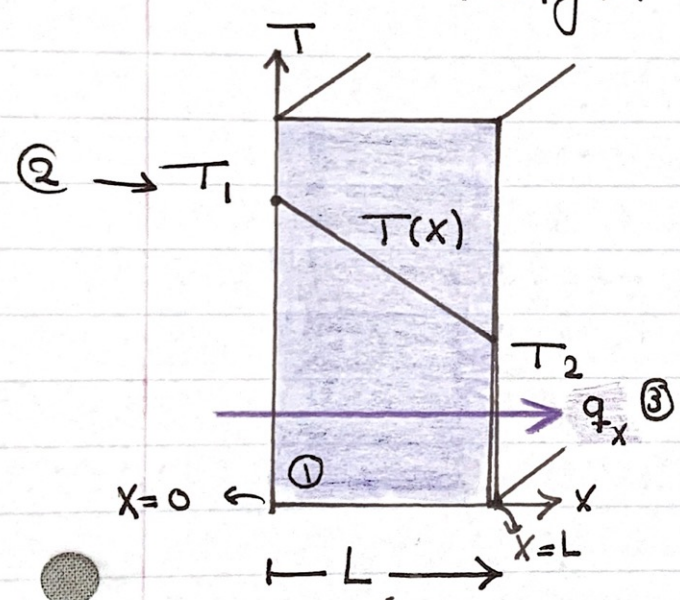
The amount of energy transferred can be computed using **rate equations**

For Conduction, this rate equation is **Fourier's law**

General form :- $q_x'' = -k \frac{dT}{dx}$

Annotations for the equation:

- q_x'' : heat flux (W/m²)
- $-k$: Thermal conductivity (W/m.K) of the material we go through
- $\frac{dT}{dx}$: In the x-direction
- Sign: Because heat goes from high to low.



- **What does this graph mean?**
- ① - solid (for example a wall)
- ② - measured temp at x=0 of the wall
- ③ - heat transfer qx

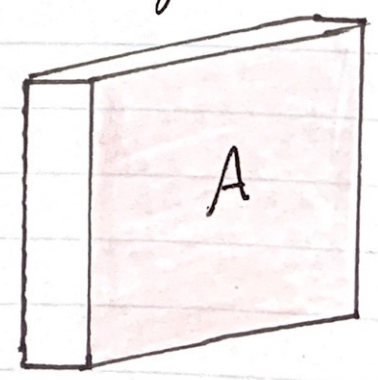
Note $q_x = q_x'' \cdot \underline{A}$

area at which heat is transferred through

Going into the law:-

$$\frac{dT}{dx} = \frac{T_2 - T_1}{L}$$

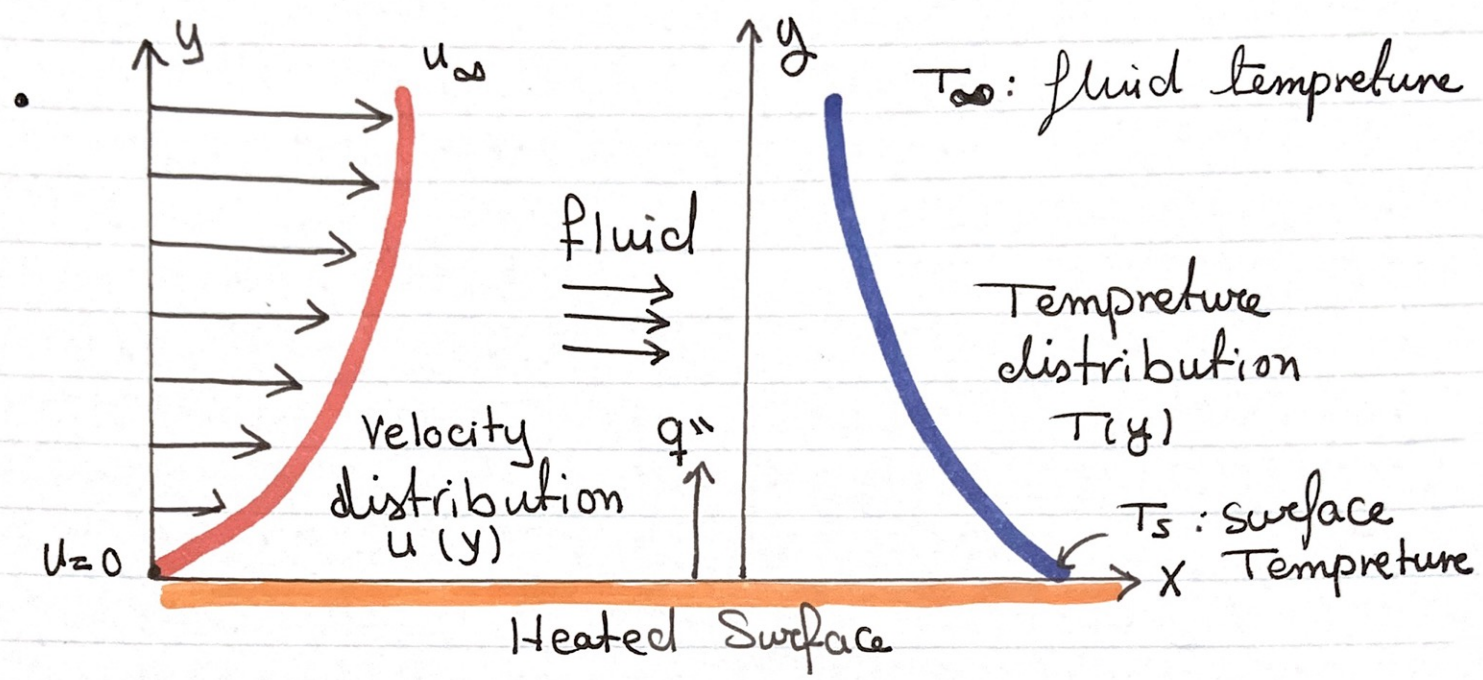
$$\text{So } q_x = -k \cdot A \left(\frac{T_2 - T_1}{L} \right)$$



2-

Convection

occurs between a surface and a moving fluid / stationary



What does this Graph mean?

- At $y=0$
 - The fluid's velocity (u) = 0, this velocity keeps rising until reaching u_{∞}
 - T_s is the temperature at $y=0$ and it is the higher temperature ($T_s > T_{\infty}$)

• So Heat transfer (convection) will occur from surface to outer flow

• The types of convection are: forced or Natural

The rate equation used for convection is:-

Newton's law of cooling

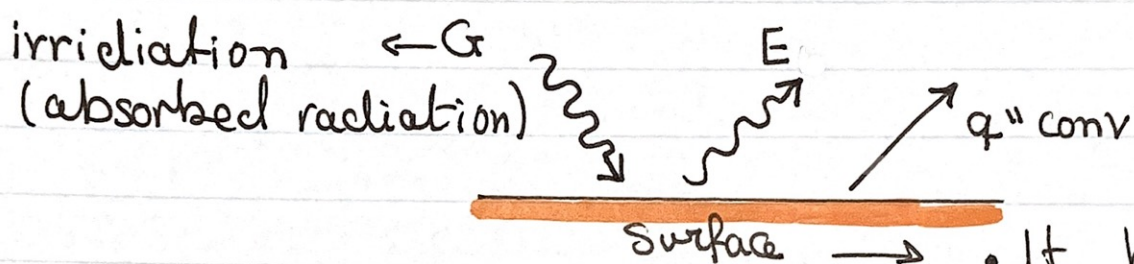
$$q'' = h (T_s - T_{\infty})$$

$$q = Ah (T_s - T_{\infty})$$

↳ convection heat transfer coefficient Depends on conditions in boundary layer

3- **Radiation** occurs as a result of electrons moving to lower energy levels

- We will focus on Radiation From Solid Surfaces
- It needs no material to be transported
It can be transported in **Vacuum**



E_b : surface emissive power
It is related to T_s

- It has temperature T_s
- emissivity ϵ , $0 \leq \epsilon \leq 1$
- absorptivity α

Stefan-Boltzmann law:-
It is equal to q''_{rad}

$$E_b = \sigma T_s^4$$

Boltzmann constant = $5.67 \times 10^{-8} \text{ W/m}^2 \cdot \text{K}^4$

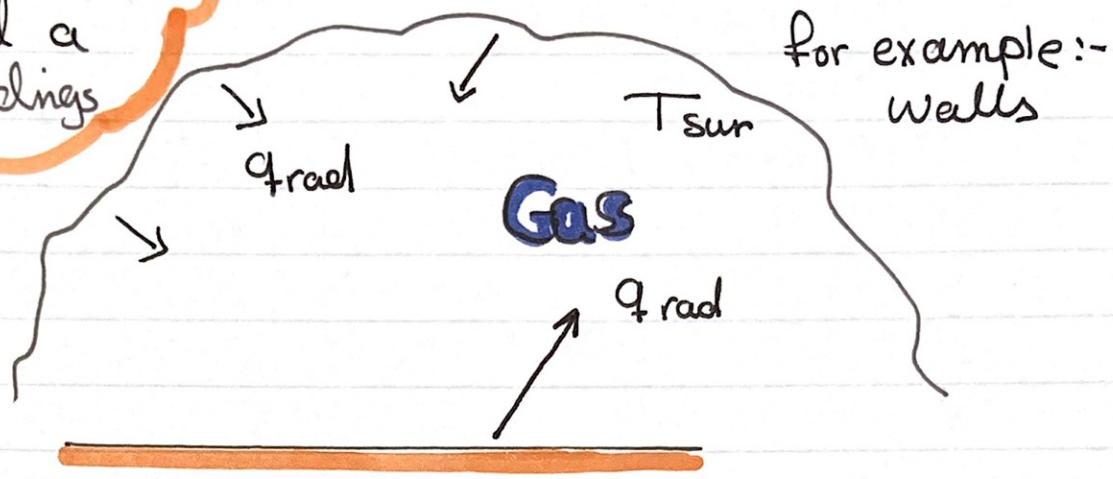
- Note: E_b is for Blackbody which is an **ideal radiator**

For a real surface:-

$$q'' = E = \epsilon \sigma T_s^4$$

Absorbed radiation $G_{abs} = \alpha G_{ir}$ ← incident radiation (irradiation) energy coming to a surface

Radiation between a surface and a large surroundings



There is an exchange between surface & surroundings

$$q = \text{Emission} - \text{Irradiation}$$

$$q = \epsilon A \sigma T_s^4 - \alpha A \sigma T_{surr}^4$$

If we have a gray surface = $\epsilon = \alpha$

$$q_{rad} = \epsilon (A \sigma) [T_s^4 - T_{surr}^4]$$

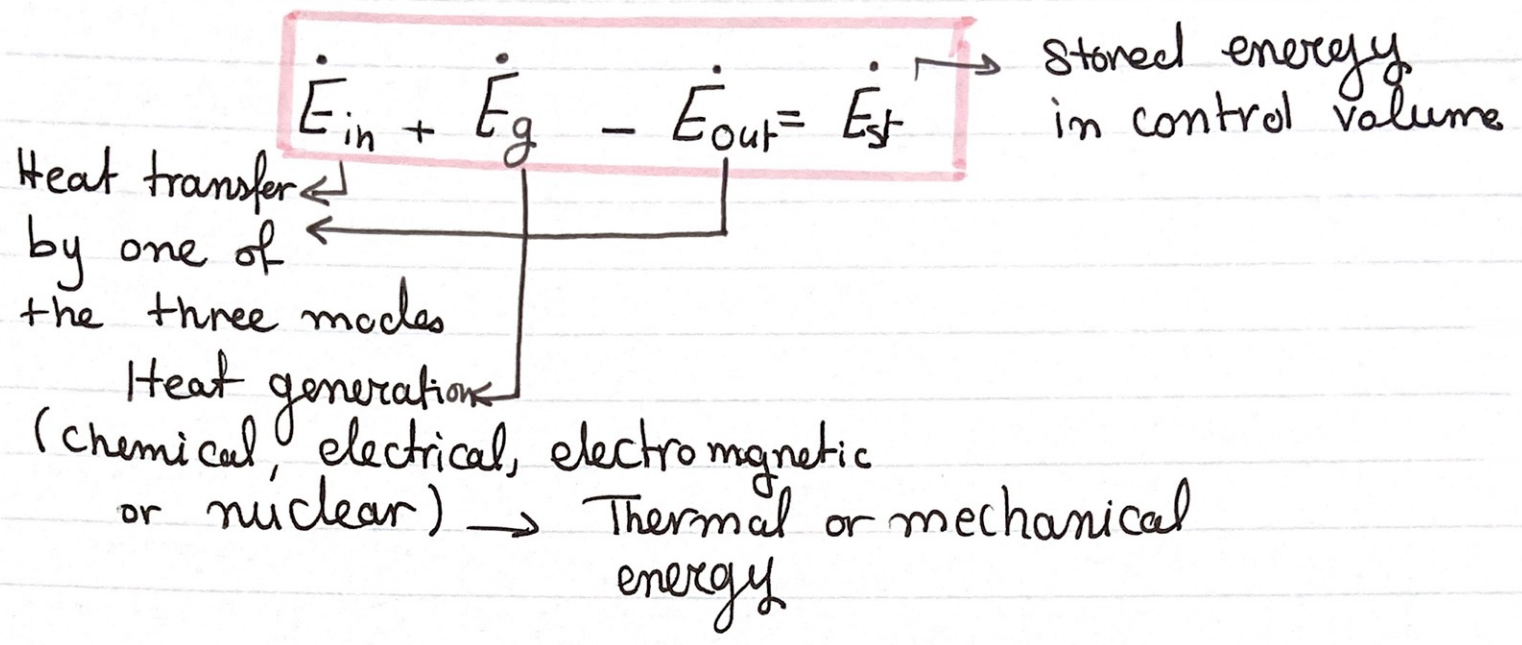
It can be simplified to

$$q_{rad} = \underline{h_r} A (T_s - T_{surr})$$

radiation heat transfer coefficient
 $h_r = \epsilon \sigma (T_s + T_{surr})(T_s^2 + T_{surr}^2)$

If we have **Gas** then (convection between Gas & surface)
 $q = q_{conv} + q_{rad}$
 $= h A (T_s - T_{\infty}) + \epsilon A \sigma (T_s^4 - T_{surr}^4)$

Conservation of energy for heat transfer



If we assumed: $\dot{E}_g = 0$, steady state: $\dot{E}_{st} = 0$
 Then

$$q = \dot{m} c_p (T_{out} - T_{in})$$

Note:
 \dot{E}_g can be electrical
 $= I^2 R_e L$

Solved Examples

1.11 Wall

(Applied) $\leftarrow q'' = 20 \text{ W/m}^2 \rightarrow$ one face
 $T = 30^\circ$ for air \rightarrow opposite face ($T = 50^\circ$)
 $k = 20 \text{ W/m}^2 \cdot \text{K}$

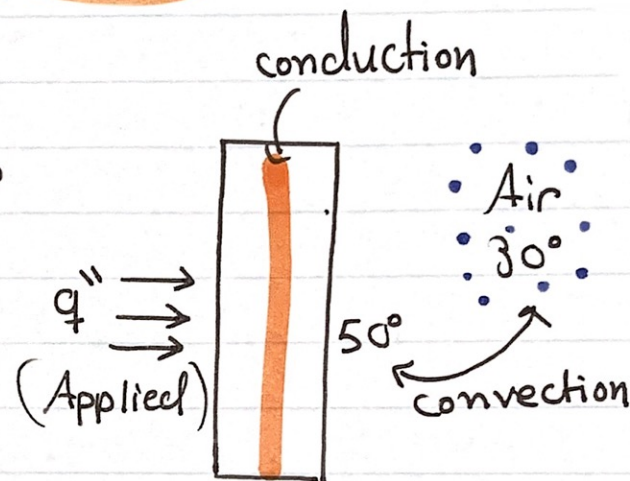
Answer : Steady state $\rightarrow \dot{E}_{st} = 0$

$$\dot{E}_{in} + \cancel{\dot{E}_g} - \dot{E}_{out} = \dot{E}_{st}$$

$$q'' A - h A (50 - 30) = ?$$

$$A [20 - \underset{20}{h} (20)] = P$$

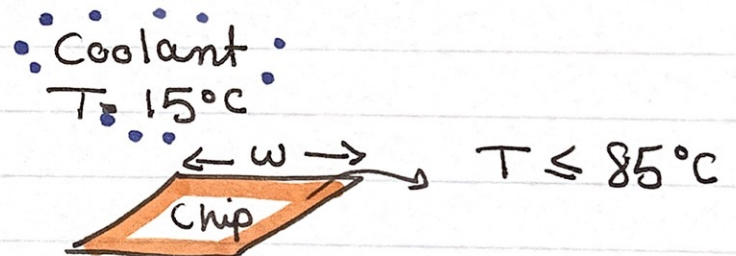
It does not exist



$\dot{E}_{st} < 0$ So:- stored energy decreasing
 So temp \downarrow

1.26

Coolant = air
 $h_1 = 200 \text{ W/m}^2 \cdot \text{K}$
 Max Chip Power?



if coolant dielectric, $h_2 = 3000 \rightarrow$ Max $P = ?$

$$q_{conv} = h_1 A (T_s - T_{\infty}) = (200) (5 \times 5 \times 10^{-6}) (85 - 15)$$

$$= 0.35 \text{ W}$$

for h_2

$$q_{conv} = (3000) (25 \times 10^{-6}) (70) = 5.25 \text{ W}$$

1.85

$G = 700 \text{ W/m}^2$

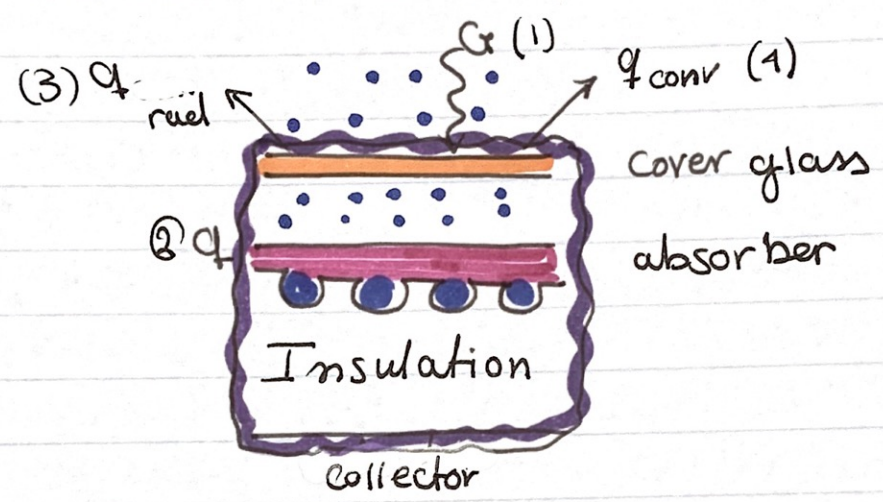
$A = 3 \text{ m}^2$

90% of solar radiation passes through glass

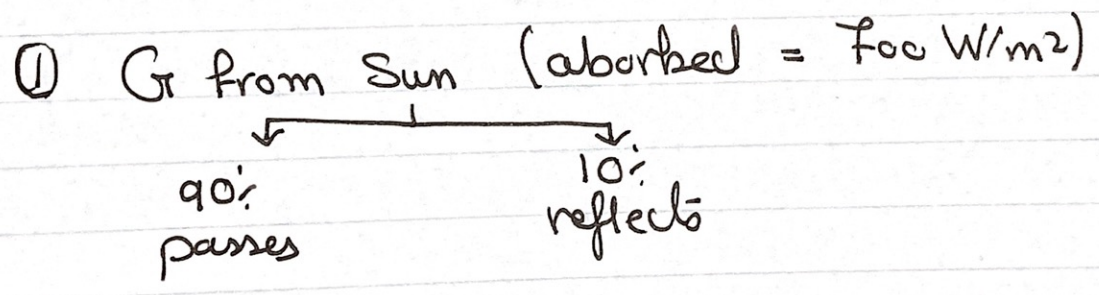
10% reflected

Water : $T_i \rightarrow T_o$

$T_{\text{cover}} = 30^\circ\text{C}$, $\epsilon = 0.91$, radiation with sky at 10°C
Convection between air & cover, $h = 10$, $T_{\text{air}} = 25^\circ$



Heat transfer we have



2) Convection between water & Absorber

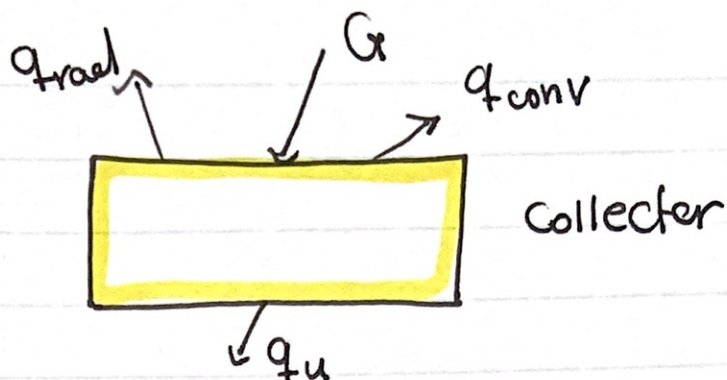
3) Radiation from cover at $T = 10^\circ$, $T_{\text{cover}} = 30^\circ$

4) Convection between air & cover



1.95

a)



$$\dot{E}_{in} - \dot{E}_{out} + \dot{E}_{gen} = 0$$

$$q_{solar} - q_{conv} - q_{rad} - q_u = 0$$

$$q_{solar} = 90\% q''(G)$$

$$q_{conv} = h(T_s - T_\infty) = 10(30 - 25)$$

$$q_{rad} = \epsilon \sigma (T_{cover}^4 - T_{sky}^4) = (0.94)(\sigma) \left((30 + 273)^4 - (10 + 273)^4 \right)$$

$$= 90\% (700) - 50 - (0.94)(5.67 \times 10^{-8})(3644541920)$$

$$= 385.75$$

b) $T_o - T_i$?
 $\dot{m} = 0.01$
 $C_p = 4179$

• Water enters collector at T_i goes out at T_o
 heat transferred = q_u

$$q = A \cdot q''_u = \dot{m} C_p (T_o - T_i)$$

$$T_o - T_i = \frac{(386)(3)}{(0.01)(4179)} = 27.7^\circ C$$

c) $\eta = \frac{q_u}{q_{solar}} = \frac{386}{700} = 0.551 = 55.1\%$